

## SEMINAR

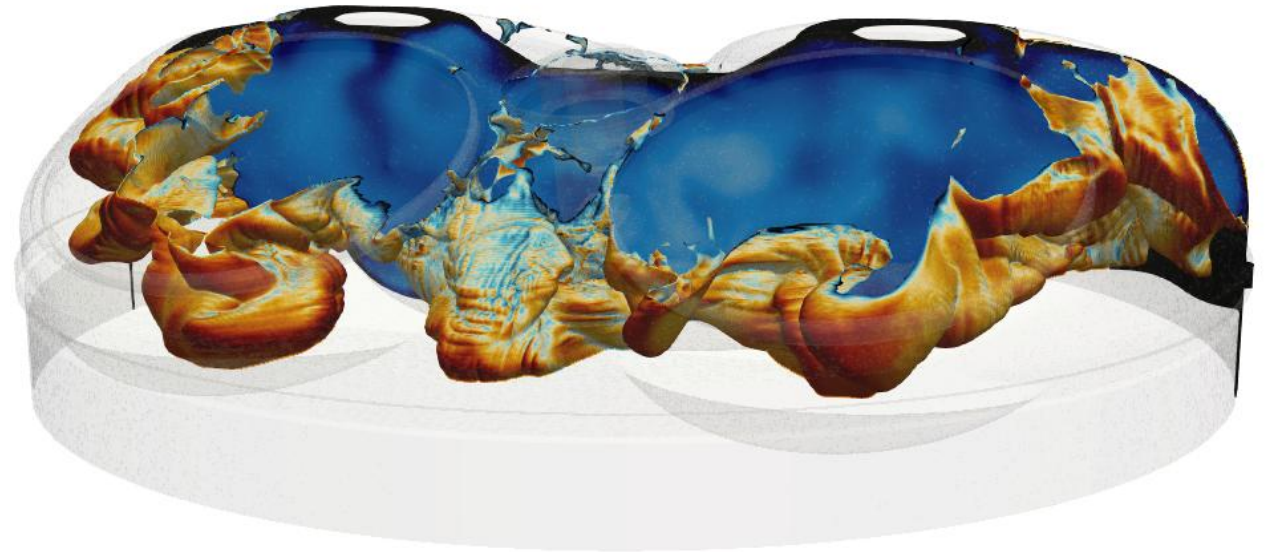
### CFD SIMULATION OF REACTIVE FLOWS: BRIDGING FUNDAMENTAL RESEARCH AND INDUSTRIAL APPLICATIONS

J. Gómez Soriano (jogoso1@mot.upv.es)

CMT – Clean Mobility & Thermofluids - Universitat Politècnica de València

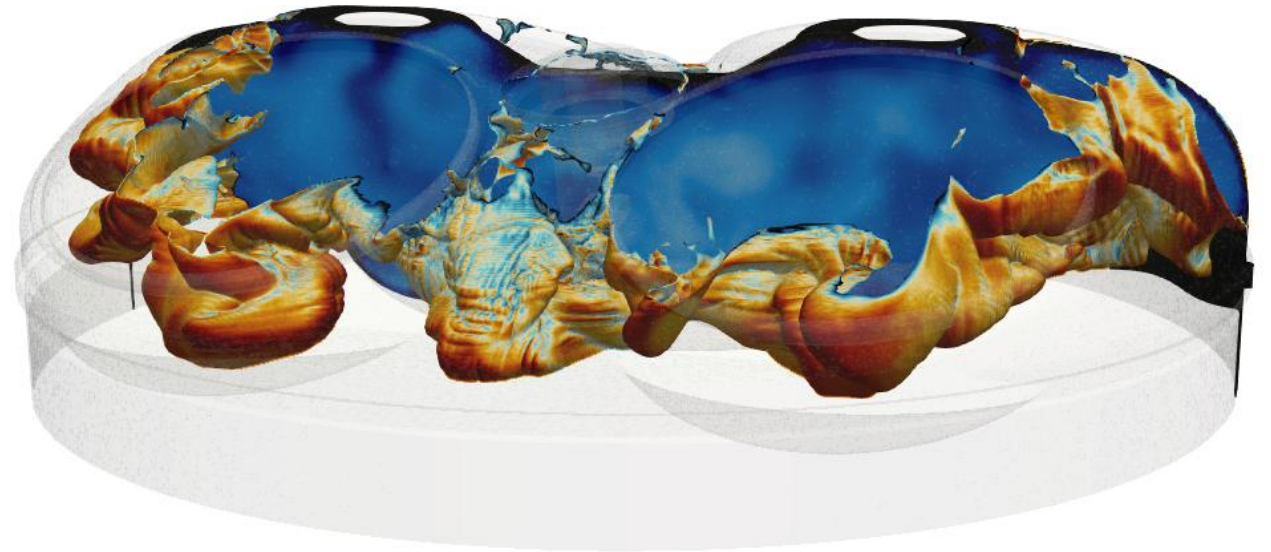
# CONTENTS

- **Introduction**
- **CFD for fundamental understanding**
  - DNS of unstable hydrogen flames
  - URANS of hydrogen combustion
  - URANS/LES of hydrogen injection
  - LES for combustion noise source characterization
- **CFD for applied research**
  - LES for optimization of pre-chamber ignition systems
  - URANS coupled with optimization techniques



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  - LES for optimization of pre-chamber ignition systems
  - URANS coupled with optimization techniques



# INTRODUCTION



UNIVERSITAT POLITÈCNICA DE VALÈNCIA

## UPV FIGURES



**Divisions:**  
14 Technical Faculties | 42 Departments | 20 Research Institutes



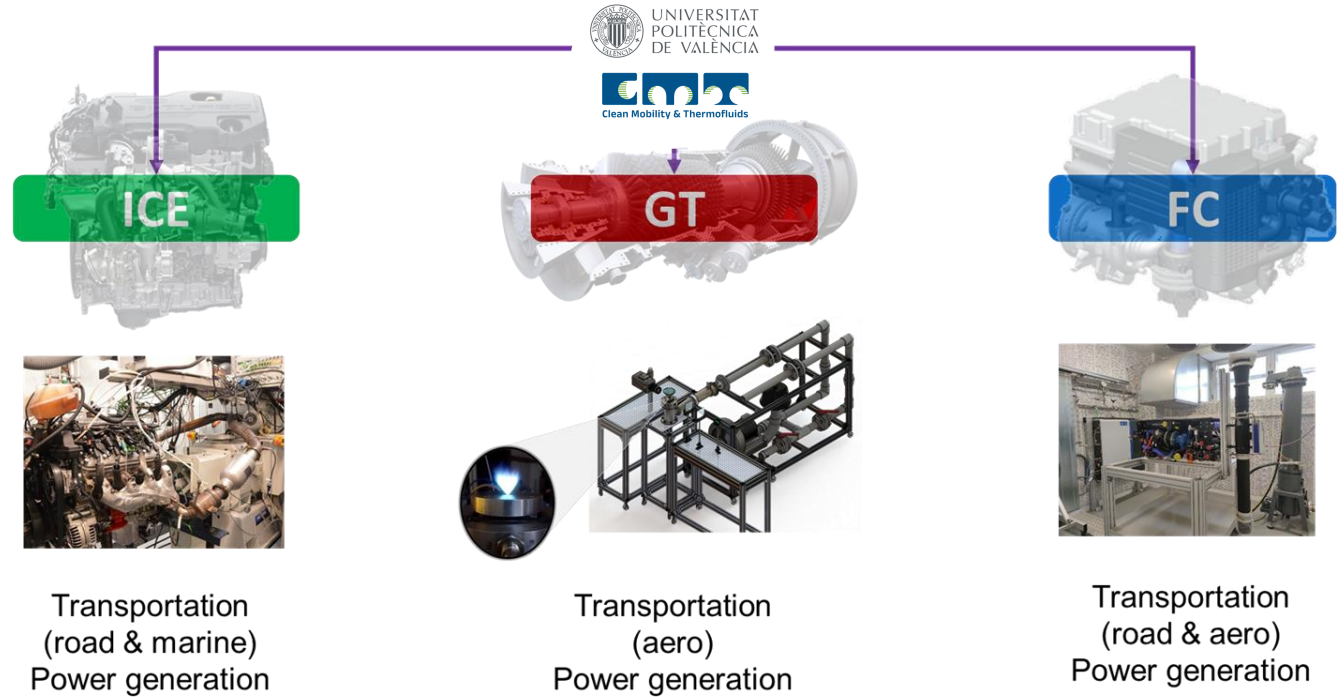
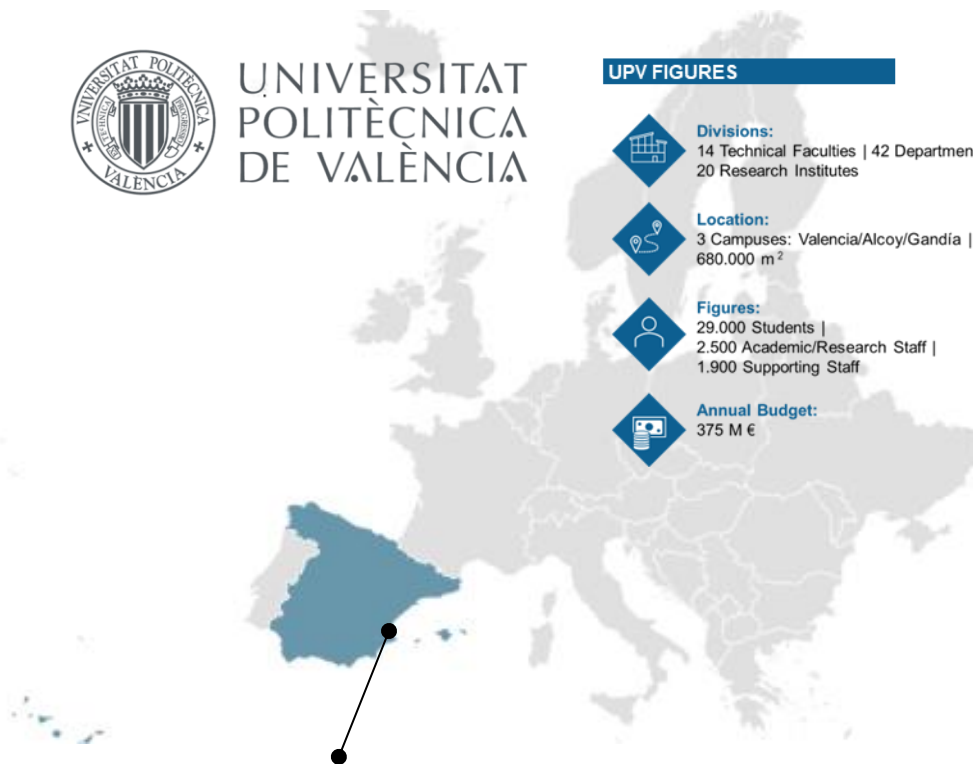
**Location:**  
3 Campuses: Valencia/Alcoy/Gandia | 680.000 m<sup>2</sup>



**Figures:**  
29.000 Students | 2.500 Academic/Research Staff | 1.900 Supporting Staff



**Annual Budget:**  
375 M €



## STAFF



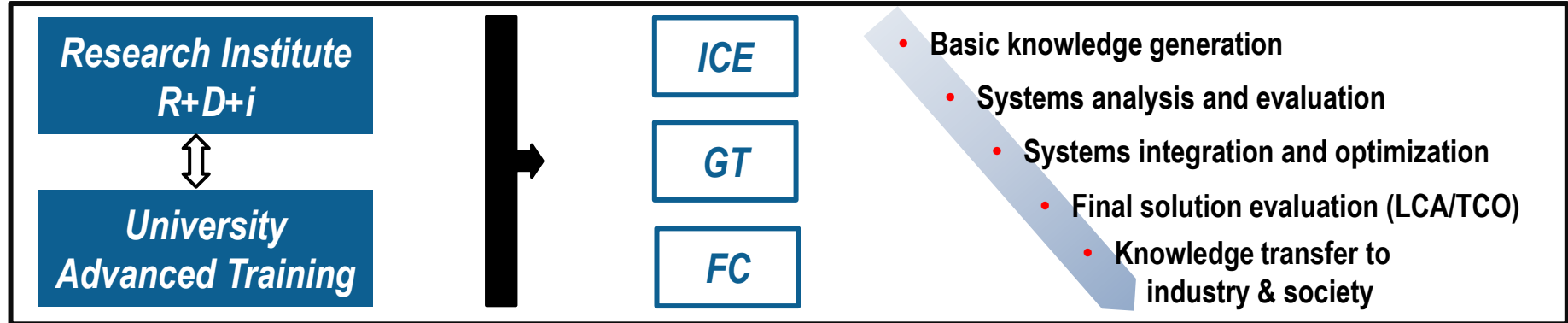
60  
Faculty / Researchers



25  
Testing Engineers

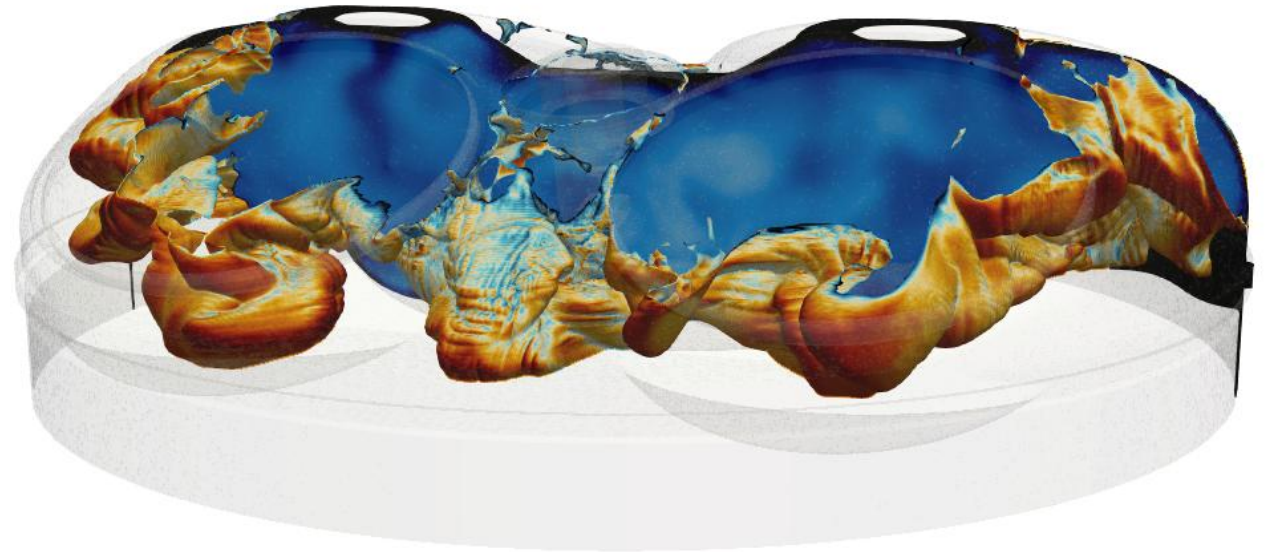


70  
MSc and PhD Students



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  - URANS coupled with optimization techniques



# CFD FOR FUNDAMENTAL UNDERSTANDING

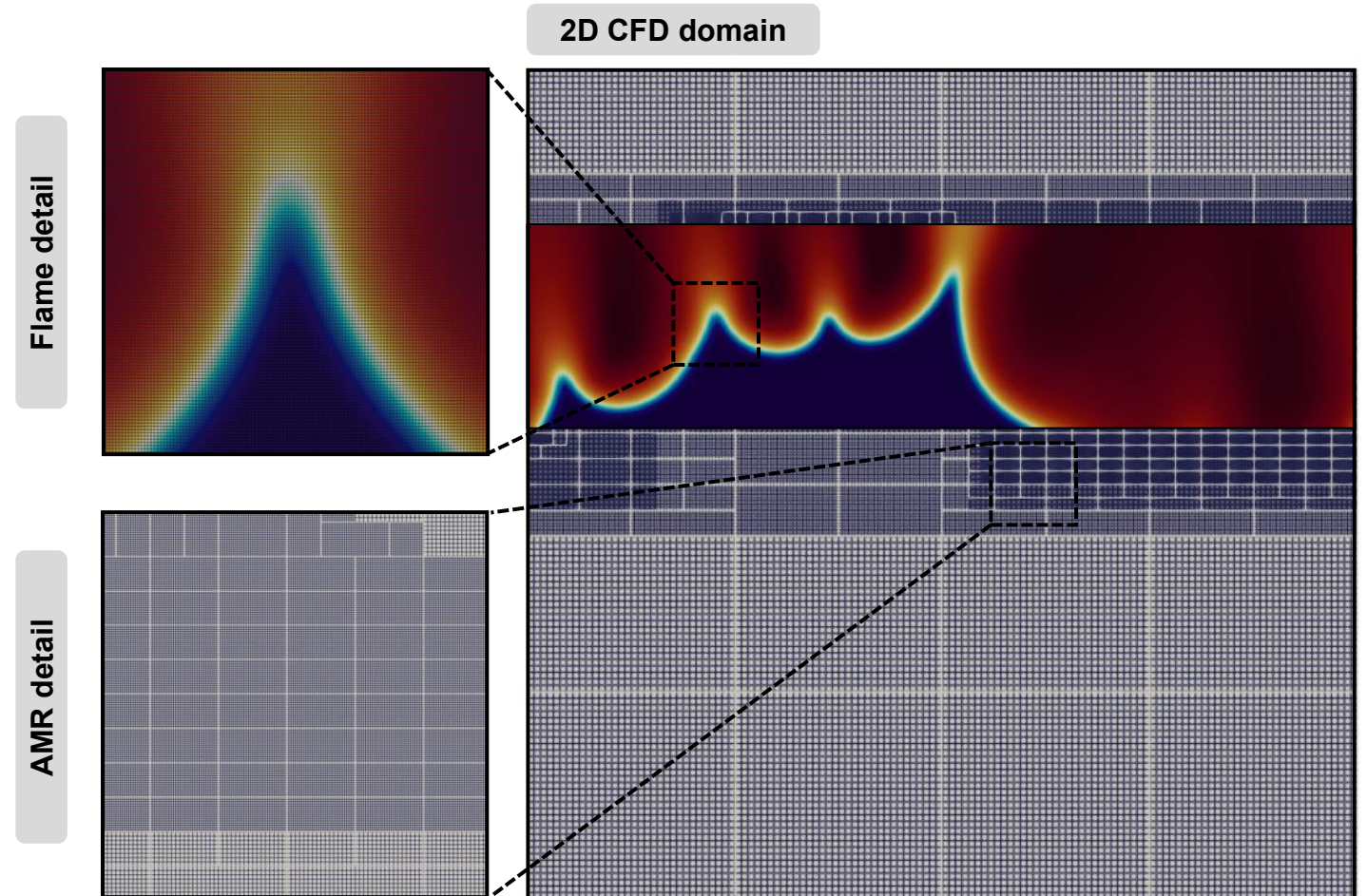
## DNS of unstable hydrogen flames

### ■ Problem definition

- Lean hydrogen flame
  - $T_u = 298$  K
  - $p = 1$  bar
  - $\phi = 0.5$
- Lewis number ( $\alpha/D$ )  $< 1$
- Laminar conditions ( $Re \downarrow \downarrow$ )

### ■ Case setup

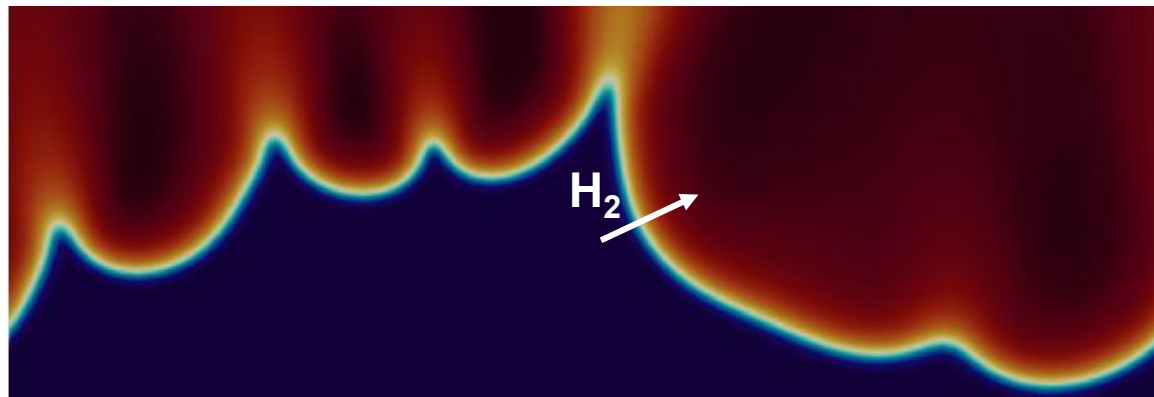
- 2D DNS with AMR
- Detailed chemical kinetics
- Domain size:
  - $60 \times l_F$  (18x18 mm)
  - Base cell size:  $140 \mu\text{m}$
  - $\sim 1$  million total cell count
- Minimum cell size:
  - $17.5 \mu\text{m}$
  - $\sim 25$  cells within thermal flame thickness ( $l_F$ )



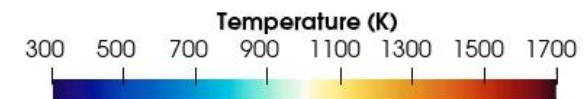
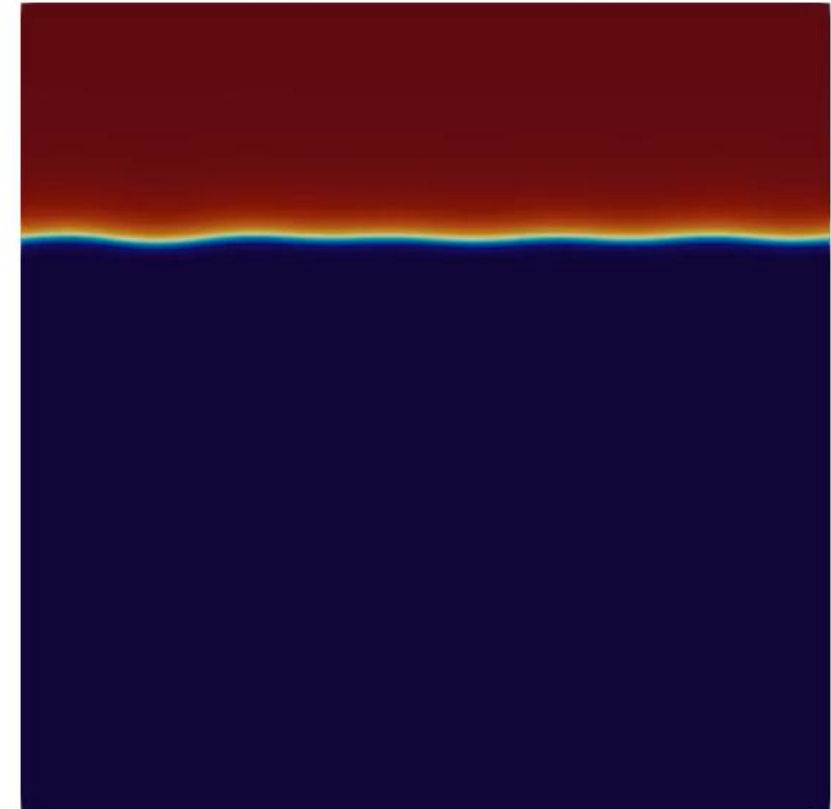
# CFD FOR FUNDAMENTAL UNDERSTANDING

## DNS of unstable hydrogen flames

- HPC resources
  - UPV cluster (Sirius)
  - GPU (NVIDIA HGX H100)
  - Return time: ~2 weeks
- Thermo-diffusive instabilities characterization
  - Caused by high mobility of H<sub>2</sub> molecules
  - Effective flame velocity increased ( $\sim \times 2$ )
  - Wavelength of flame perturbations ( $\sim 6 \times l_F$ )

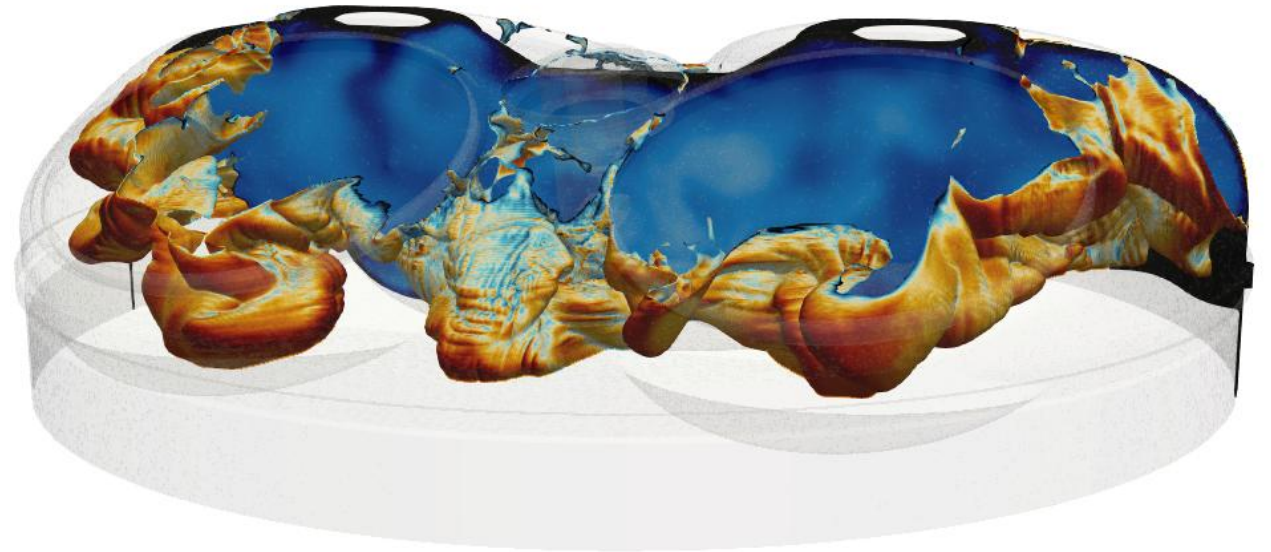


0.00 ms



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  - LES for optimization of pre-chamber ignition systems
  - URANS coupled with optimization techniques



# CFD FOR FUNDAMENTAL UNDERSTANDING

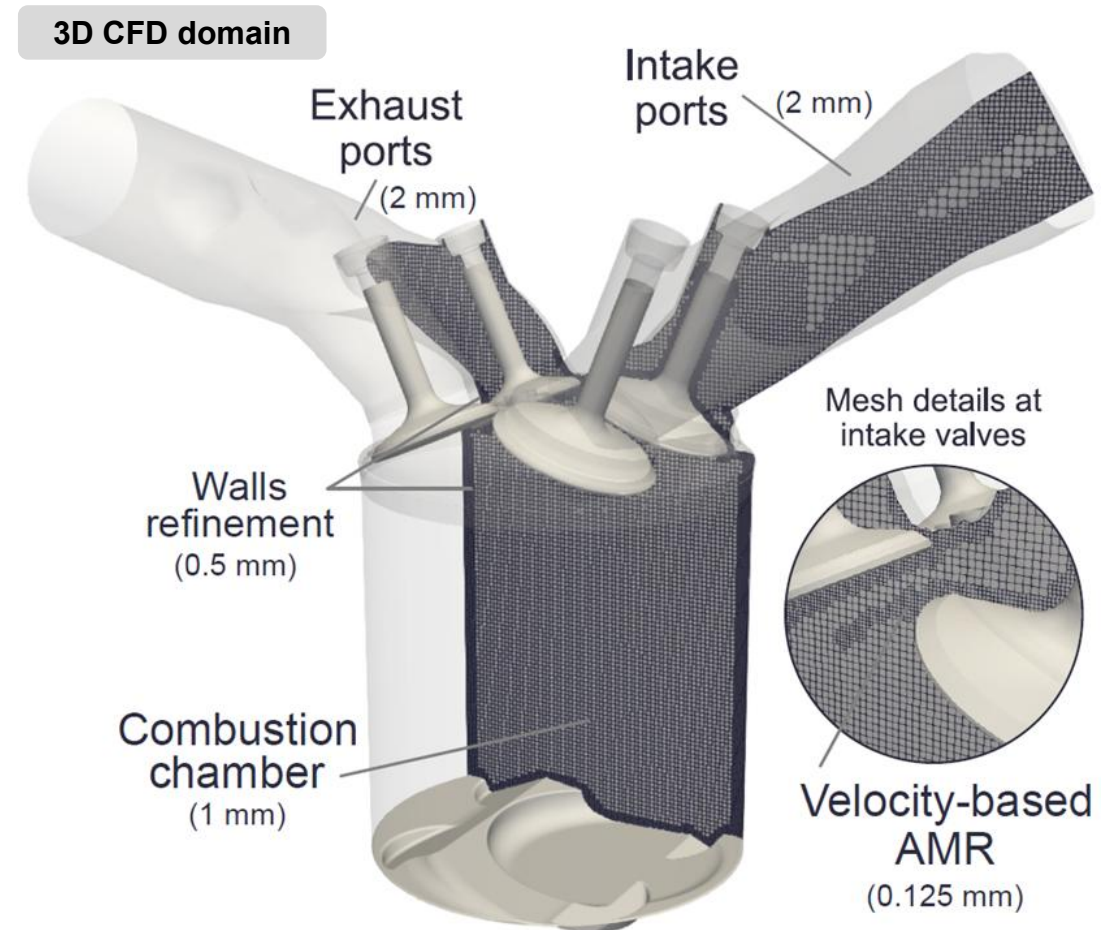
## URANS of hydrogen combustion: from DNS to URANS models

### ■ Thermo-diffusive instabilities characterization from DNS

- New H<sub>2</sub> combustion models: development and implementation

### ■ Numerical model setup

- CONVERGE v3.0.28
- Turbulence modeling: URANS, RNG k- $\epsilon$
- Cell sizes
  - AMR: Velocity (0.125 mm), Temperature (0.125 mm)
  - Spark embedding: 0.0625 mm around the spark
  - Total cell count: varied between 1 and 5 million
- Specific sub-models/features
  - G-Equation + SAGE (in/on/out of the flame)
  - Tabulated laminar flame speed ( $s_L$ )
    - Corrected with **thermo-diffusive instabilities effects**
  - Chemical mechanism
    - Kéromnès et al. (10.1016/j.combustflame.2013.01.001)
  - Molecular diffusion modeling
    - Mixture averaged



# CFD FOR FUNDAMENTAL UNDERSTANDING

## URANS of hydrogen combustion: from DNS to URANS models

- Thermo-diffusive instabilities characterization from DNS
  - New H<sub>2</sub> combustion models: development and implementation

$$\frac{\partial \rho \tilde{G}}{\partial t} + \nabla(\rho \tilde{u} \tilde{G}) = -\rho D_t \tilde{\kappa} |\nabla \tilde{G}| + \rho (s_T) \nabla \tilde{G}$$

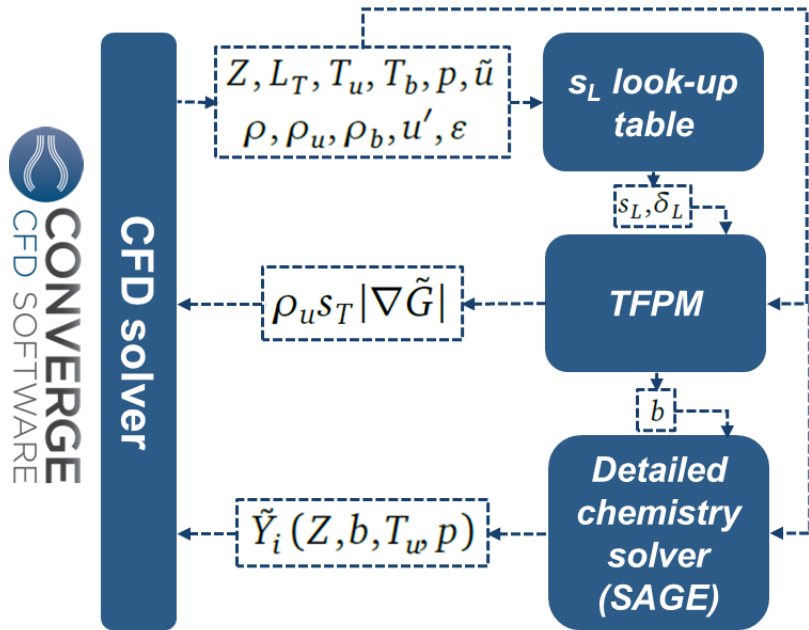
### Turbulent flame propagation

Peters' correlation

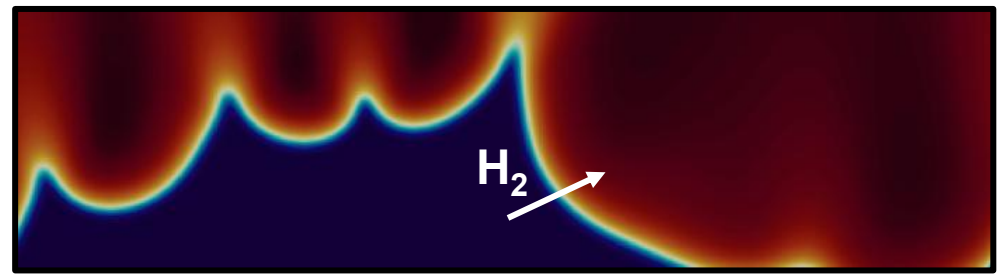
$$\frac{s_T}{s_L} = 1 - \frac{a_4 b_3^2 L_T}{2b_1 \delta_L} + \left[ \left( \frac{a_4 b_3^2 L_T}{2b_1 \delta_L} \right)^2 + a_4 b_3^2 \frac{u' L_T}{s_L \delta_L} \right]^{1/2}$$

### Correction for thermo-diffusive instabilities

- A correction can be applied to the unstretched laminar flame speed



$$s_M = I_0 \cdot s_L \quad I_0 = \begin{cases} \exp(0.08\omega_2) & \text{if } \Pi < \Pi_c, \\ 1 + 0.47\omega_2 & \text{otherwise.} \end{cases}$$

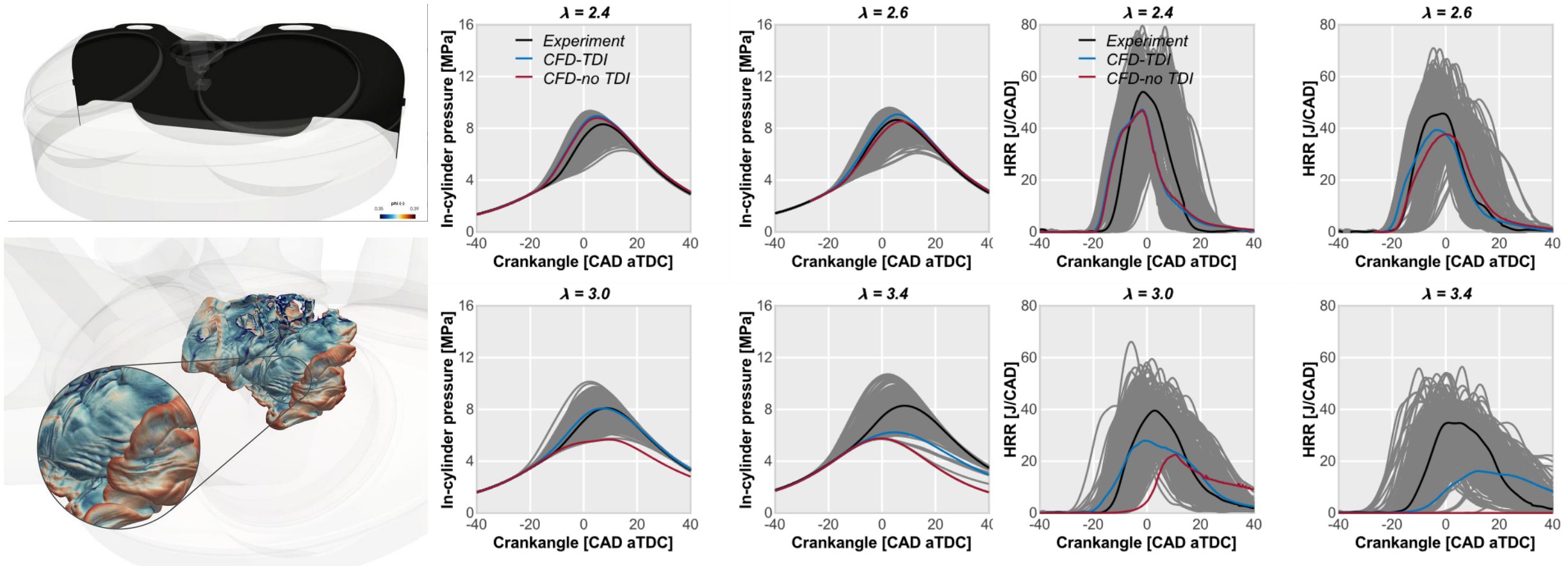


# CFD FOR FUNDAMENTAL UNDERSTANDING

## URANS of hydrogen combustion: from DNS to URANS models

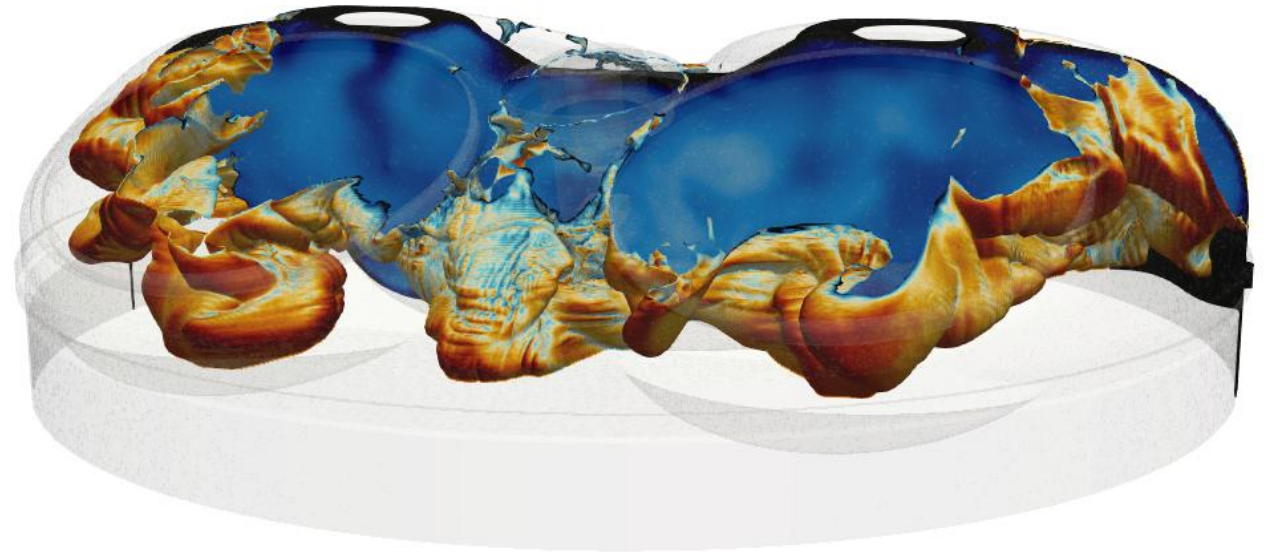
- Validation: comparison with experimental results
  - In-cylinder pressure and HRR profiles

1500 rpm @ 8.5 bar of IMEP  
4 dilution levels ( $\lambda$ )  
Calibration case:  $\lambda = 2.6$   
Same constants are applied to all OP



# CONTENTS

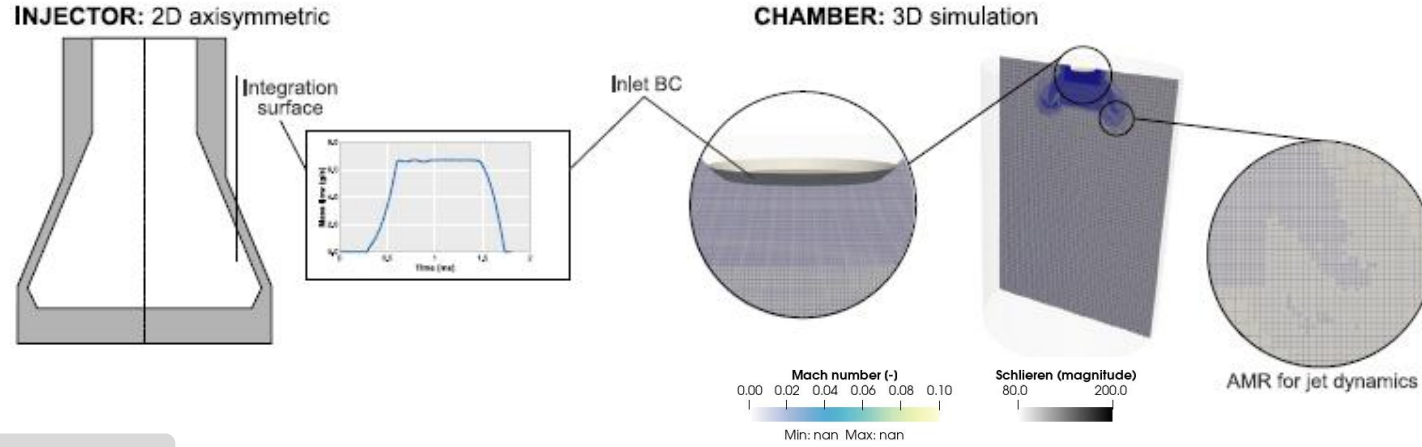
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  - LES for optimization of pre-chamber ignition systems
  - URANS coupled with optimization techniques



# CFD FOR FUNDAMENTAL UNDERSTANDING

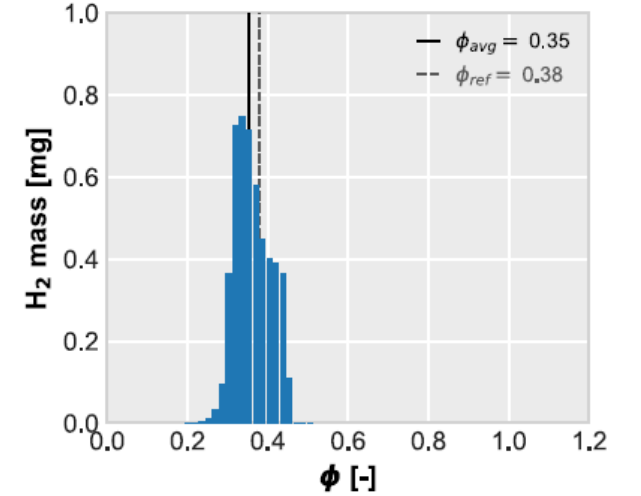
## URANS/LES of hydrogen injection

- Modeling hydrogen injection under critical conditions
  - Combination of different modeling approaches

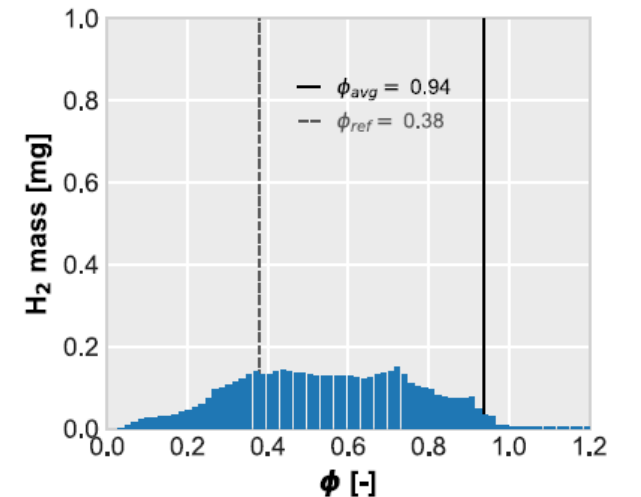


### Results & distribution analysis

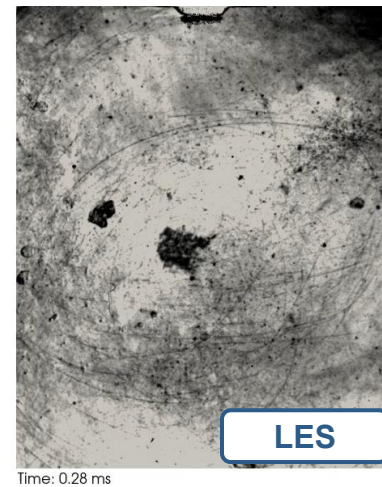
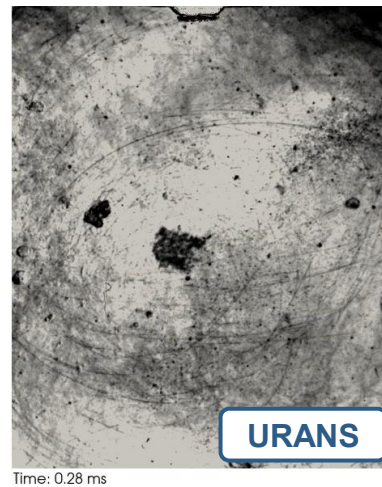
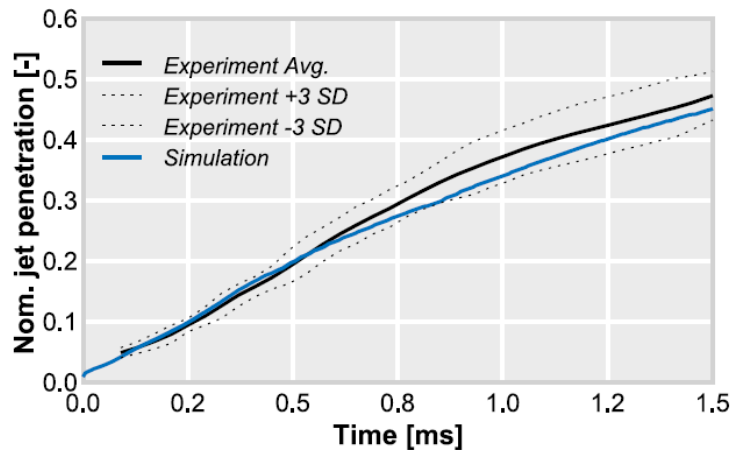
OP1 (SOI = -130 CAD aTDC,  $\lambda = 2.6$ )



OP2 (SOI = -65 CAD aTDC,  $\lambda = 2.6$ )

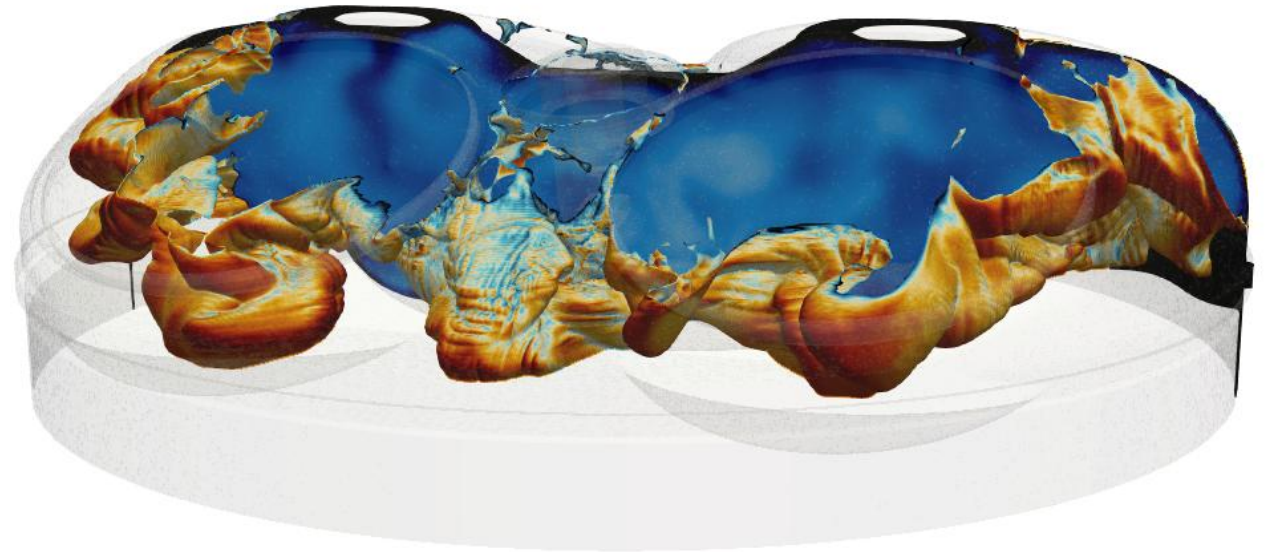


### Validation



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# CFD FOR FUNDAMENTAL UNDERSTANDING

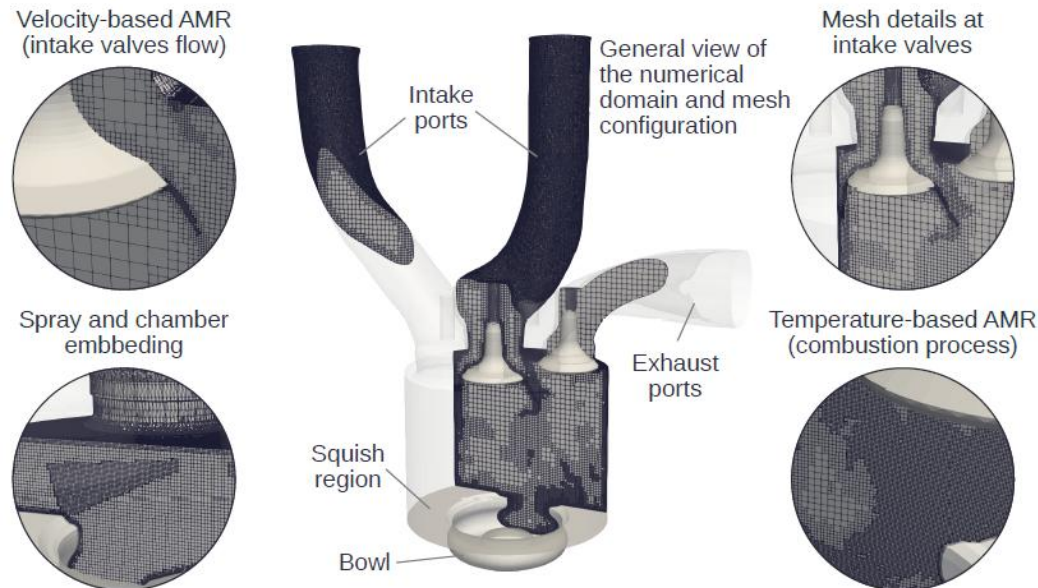
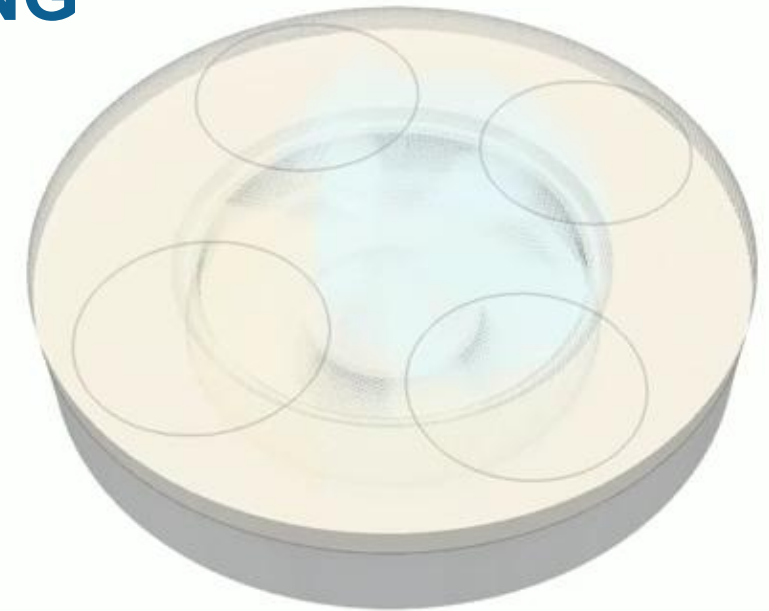
## LES for combustion noise source characterization

### ■ Problem definition

- CC resonance in CI engines
- Characterization of sources
- Assessment of simulation complexity and cost

### ■ Case setup

- 3D LES/URANS with AMR
- Detailed chemical kinetics

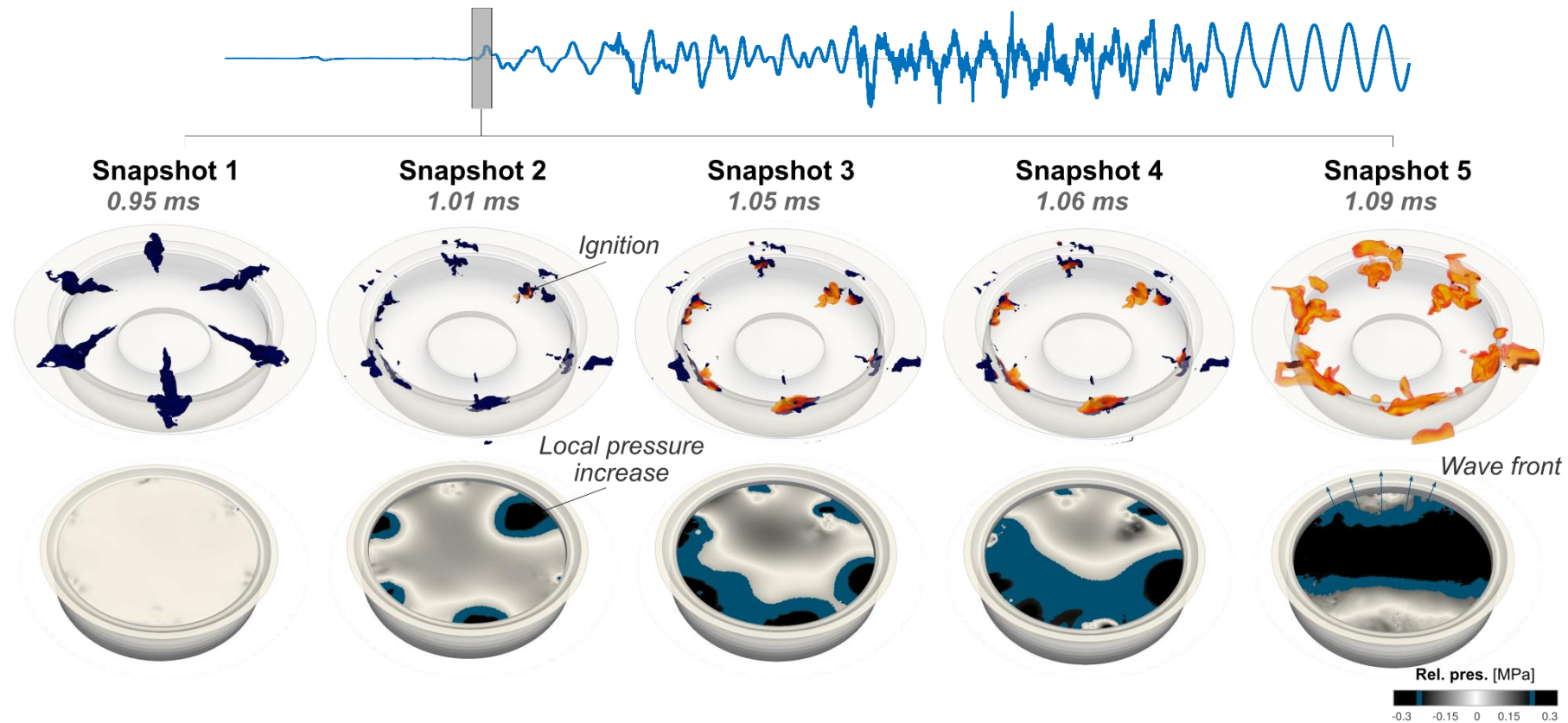


Configuration	URANS-Coarse <sup>a</sup>	URANS-Fine	LES
Domain		Full geometry <sup>b</sup>	
Base size	3 mm	3 mm	2.2 mm
Walls refinement	3 (0.375 mm)	3 (0.375 mm)	3 (0.275 mm)
Spray refinement	3 (0.375 mm)	3 (0.375 mm)	3 (0.275 mm)
Chamber refinement	2 (0.75 mm)	2 (0.75 mm)	2 (0.55 mm)
AMR	3 (0.375 mm)	4 (0.188 mm)	4 (0.138 mm)
Sonic CFL number		< 1	
Initialization		Non-uniform <sup>c</sup>	
Turbulence modelling	URANS RNG k-ε		Dynamic structure LES

# CFD FOR FUNDAMENTAL UNDERSTANDING

## LES for combustion noise source characterization

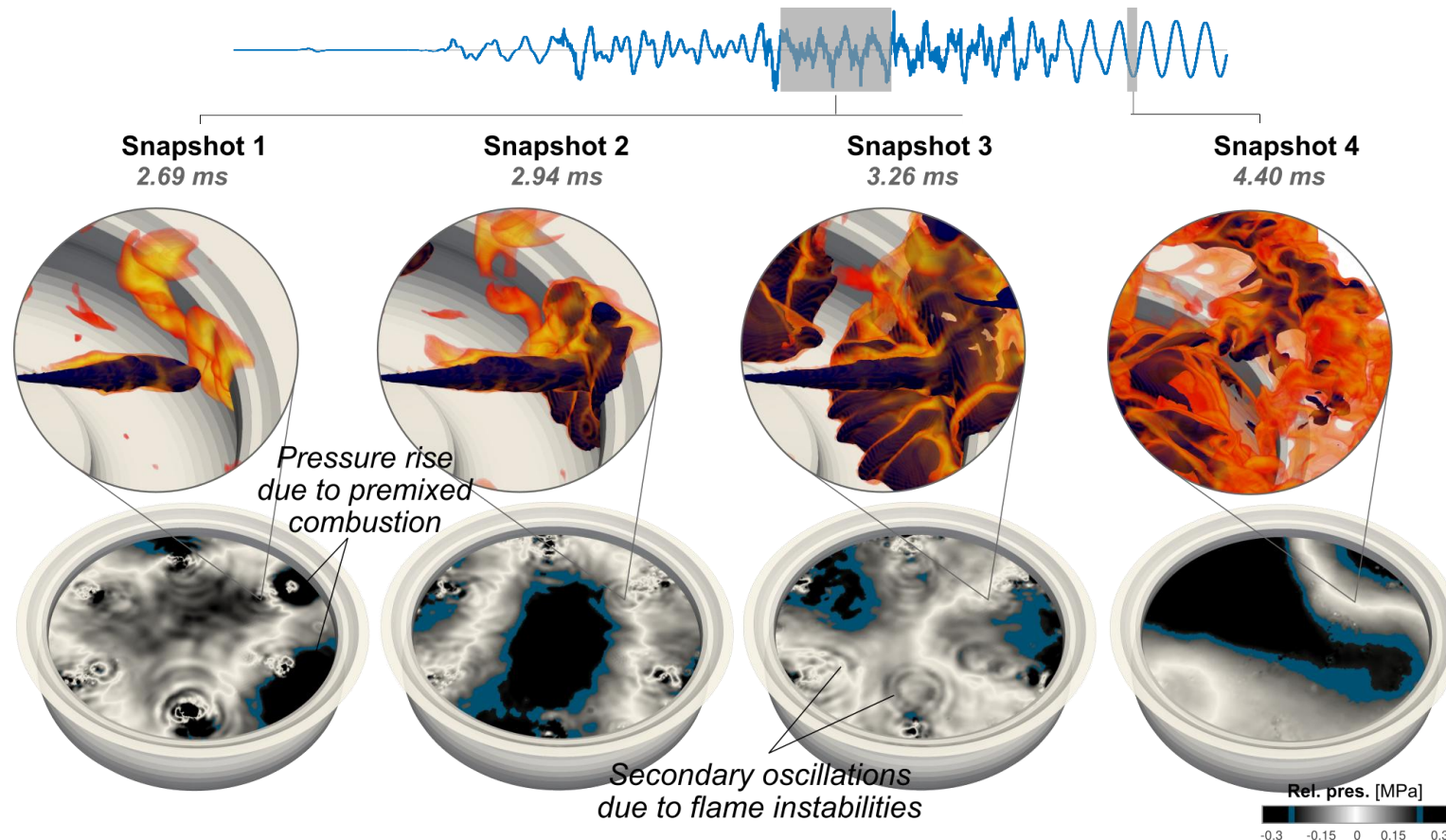
- Phenomenological analysis of combustion noise sources
  - Premixed combustion phase



# CFD FOR FUNDAMENTAL UNDERSTANDING

## LES for combustion noise source characterization

- Phenomenological analysis of combustion noise sources
  - Diffusive combustion phase



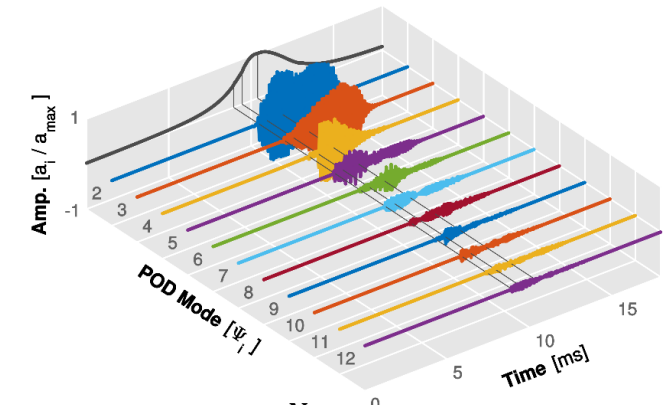
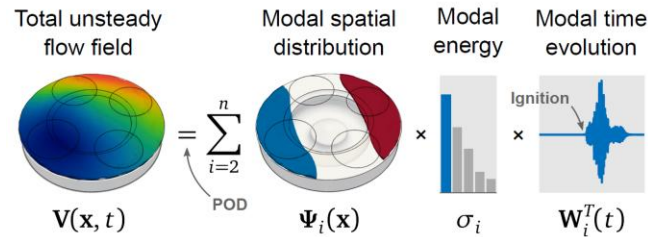
# CFD FOR FUNDAMENTAL UNDERSTANDING

## LES for combustion noise source characterization

### Phenomenological analysis of combustion noise sources

#### Modal decomposition techniques

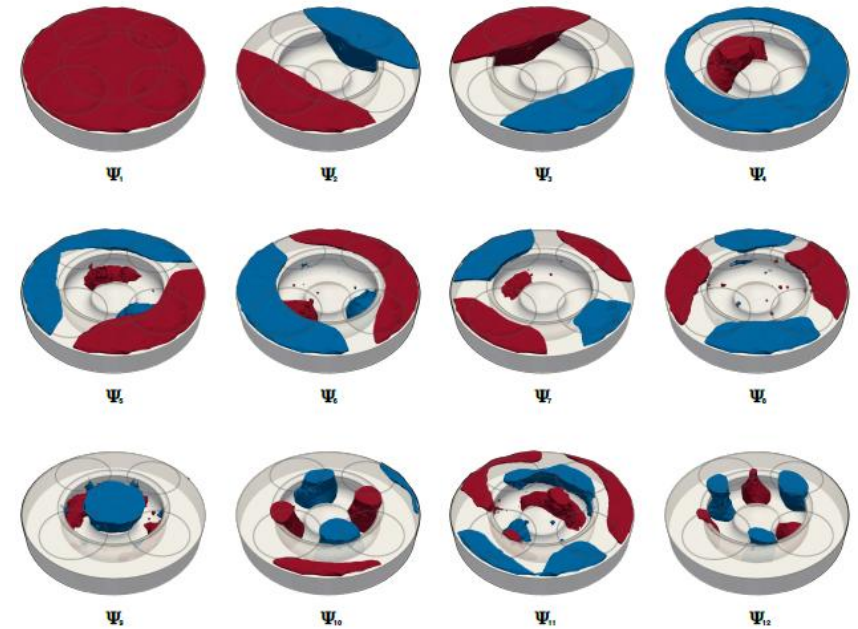
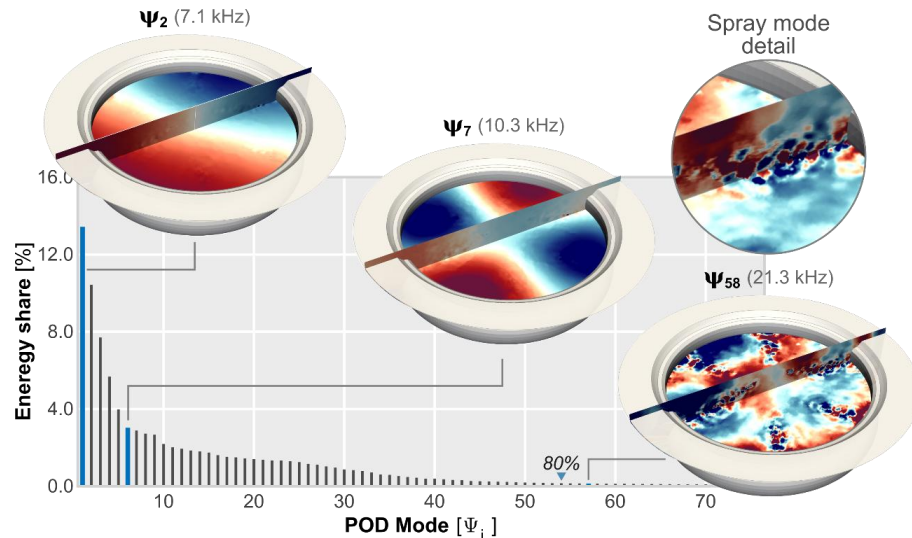
- Proper Orthogonal Decomposition
  - Spatial oscillation pattern
  - Time evolution
  - Energy contribution



$$P(x, t) \xRightarrow{SVD} \sum_{i=1}^N \Psi_i(x) \cdot \sigma_i \cdot W_i^T(t)$$

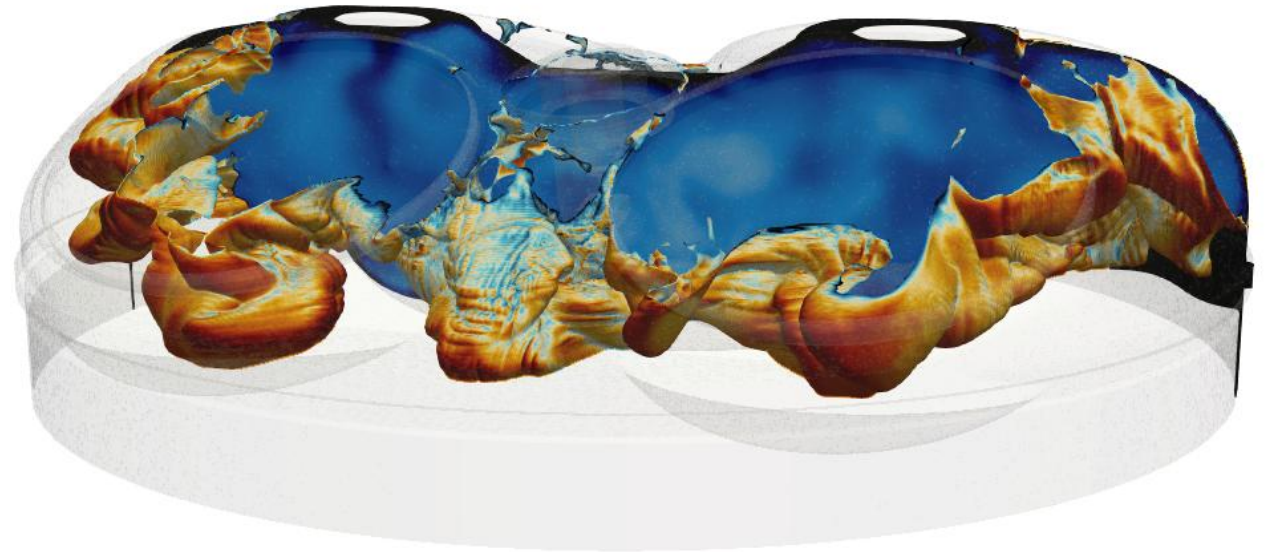
#### The size of the oscillation structures correlates with the energy contribution

- Large-scale oscillation structures gather 80% of the energy
- Jet-induced structures only constitute 20% of the energy



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# CFD FOR APPLIED RESEARCH

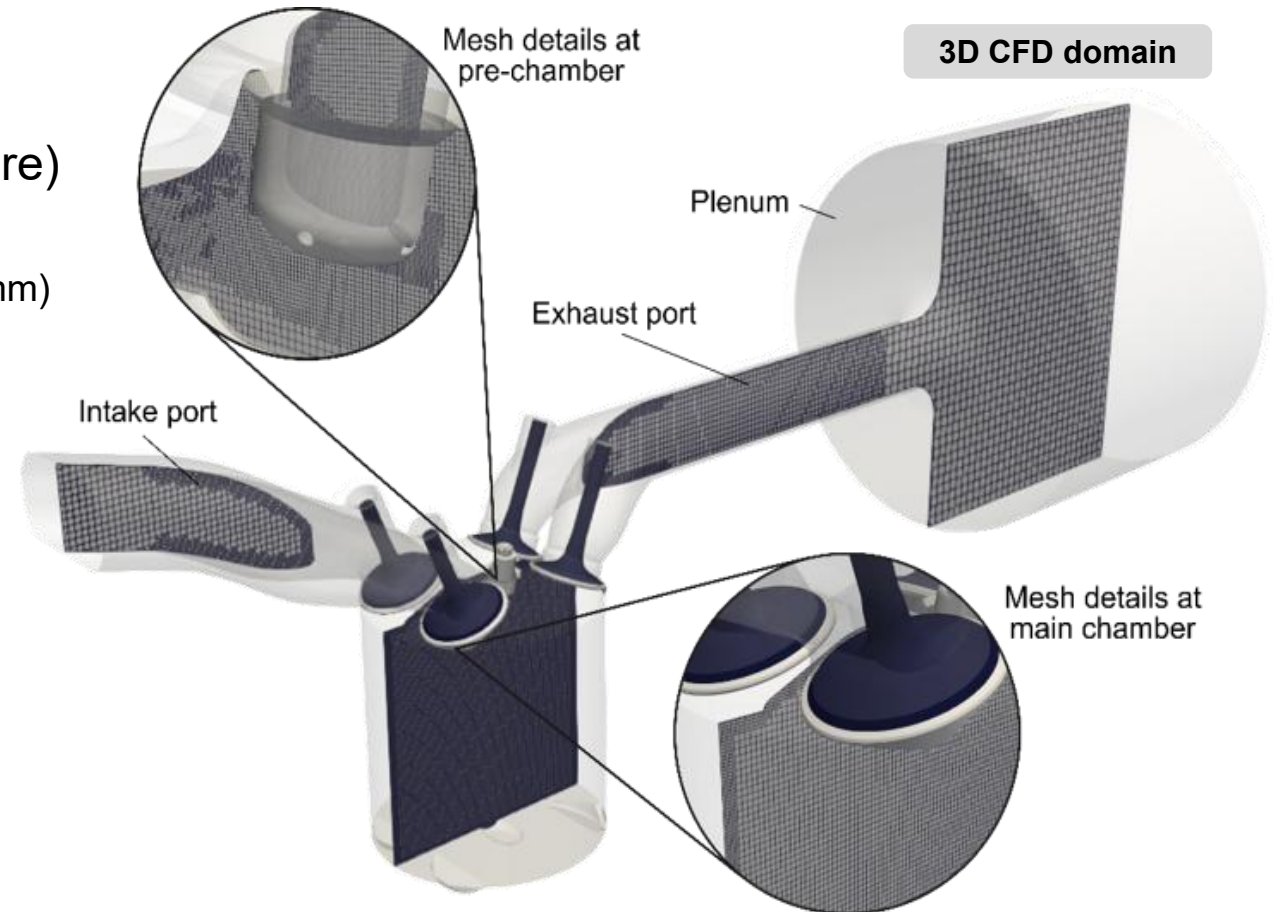
## LES for optimization of pre-chamber ignition systems

### ■ Problem definition

- Pre-chamber ignition concept modelling

### ■ Numerical model setup

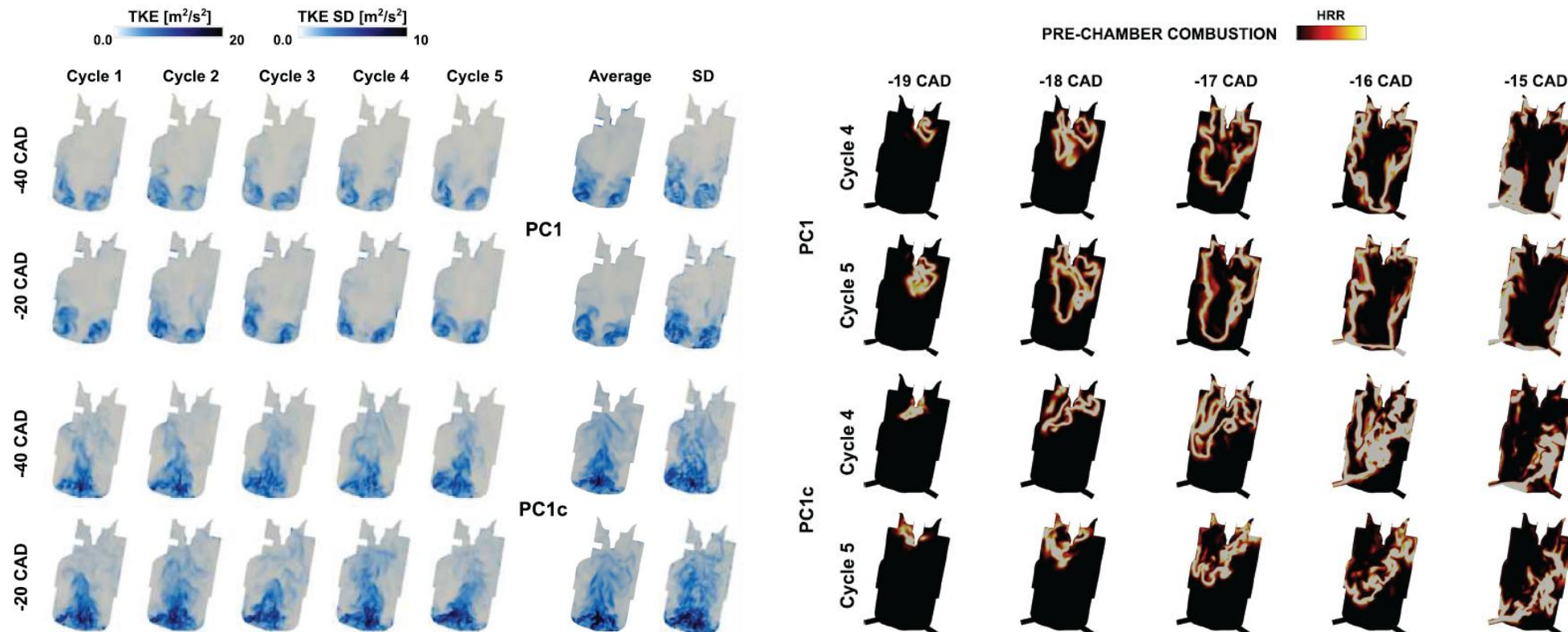
- CONVERGE v3.0.28
- Turbulence modeling: LES (dynamic structure)
- Cell sizes
  - AMR: Velocity (0.0625 mm), Temperature (0.625 mm)
  - Spark embedding: 0.0312 mm around the spark
  - Total cell count: varied between 10 and 15 million
- Specific sub-models/features
  - G-Equation + SAGE (in/on/out of the flame)
  - Tabulated laminar flame speed ( $s_L$ )
  - Chemical mechanism
    - Liu et al. (gasoline fuel)



# CFD FOR APPLIED RESEARCH

## LES for optimization of pre-chamber ignition systems

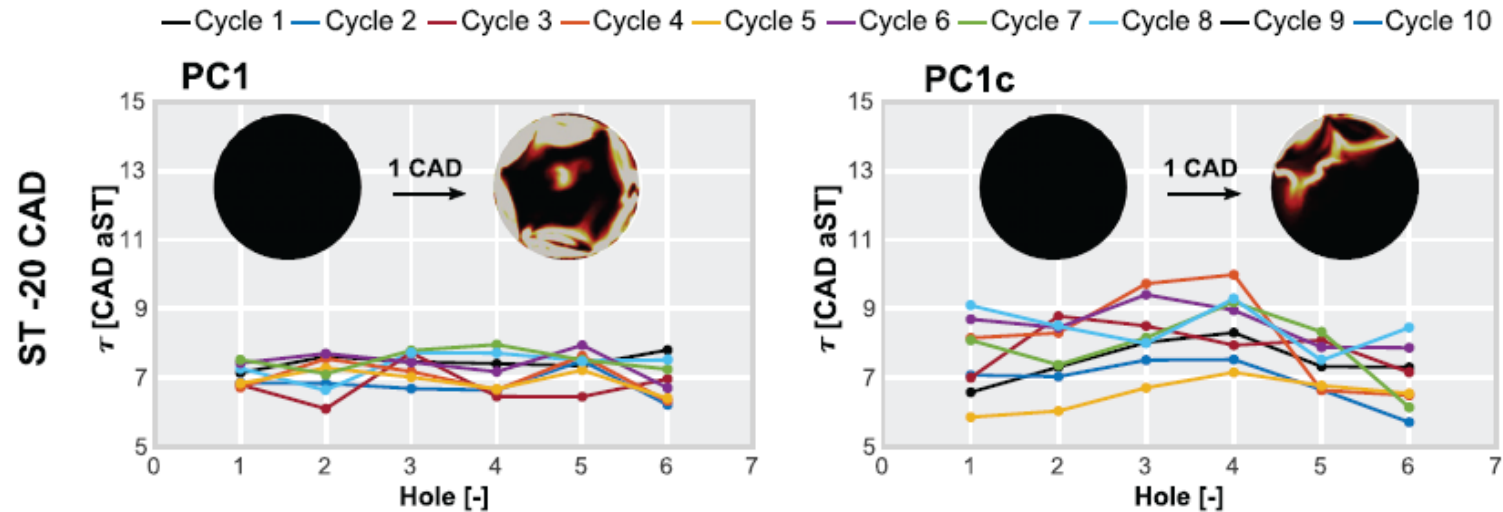
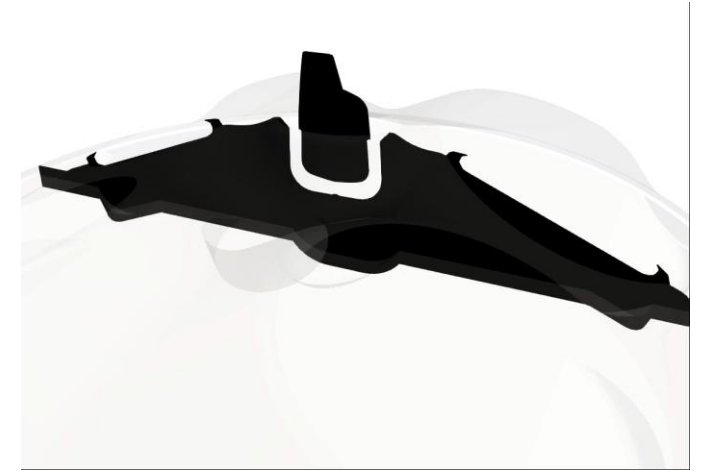
- Analysis of cycle-to-cycle variability
  - Pre-chamber flow patterns
  - Combustion progress inside the pre-chamber
  - Uniformity of flame patterns and nozzle distribution



# CFD FOR APPLIED RESEARCH

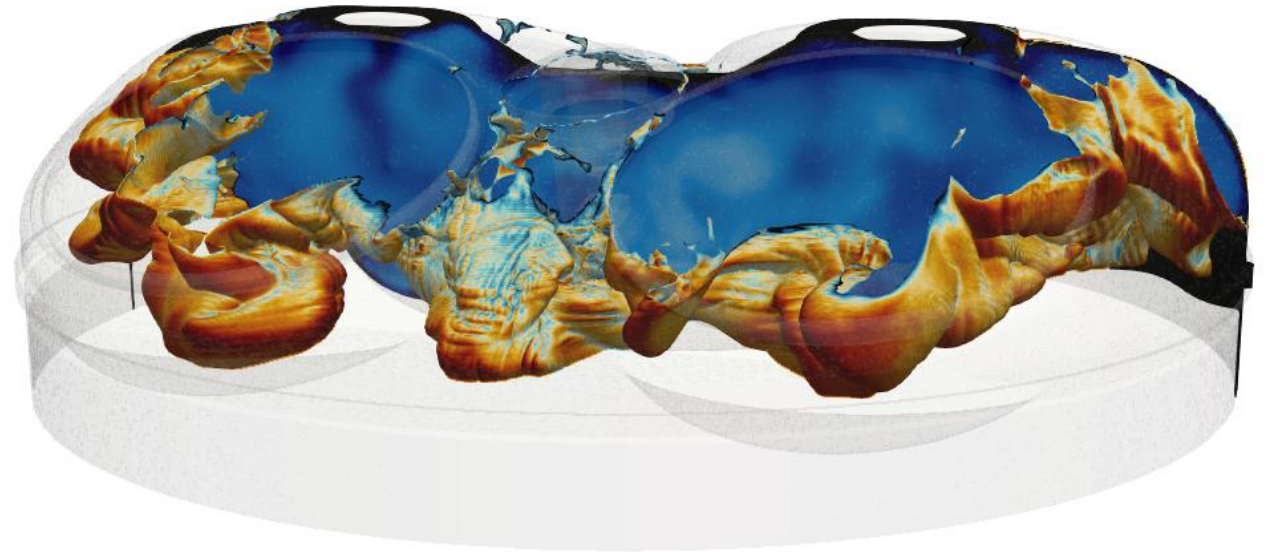
## LES for optimization of pre-chamber ignition systems

- Analysis of cycle-to-cycle variability
  - Pre-chamber flow patterns
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# CFD FOR APPLIED RESEARCH

## URANS coupled with optimization techniques

### ■ Noise optimization - GA optimization methodology

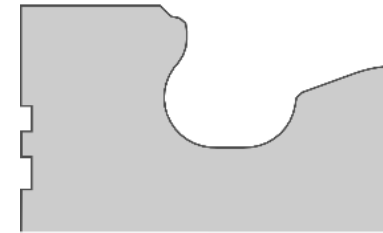
- Combustion system design
- Evolutionary optimization method (genetic algorithms)
  - **8 optimizing parameters**
    - 5 geometric parameters for piston bowl
    - Included angle, number of nozzle holes
    - 1 parameter for intake port config. (swirl)
  - **Constraints**
    - Controlled emissions: NOx and soot
  - **Objectives**
    - Minimize resonance energy → Noise ↓↓
    - Minimize ISFC → Efficiency ↑

### Ranges of the optimizing parameters

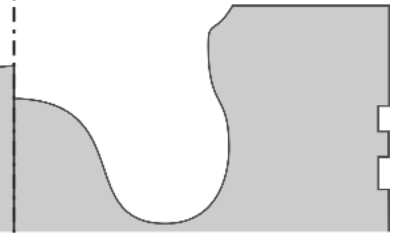
Geometric parameter 1 [-]	0.01–0.99
Geometric parameter 2 [-]	0.01–0.99
Geometric parameter 3 [-]	0.01–0.99
Geometric parameter 4 [-]	0.01–0.99
Geometric parameter 5 [-]	0.01–0.99
Number of injector nozzles [-]	4–12
Spray included angle [-]	80–180
Swirl number at IVC [-]	0.0–2.0

### Example of different bowl profiles obtained by the Bezier polynomial method

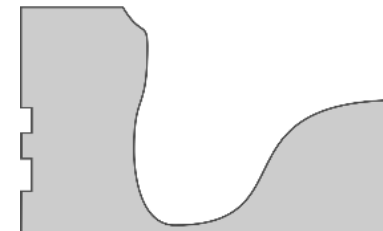
Baseline geometry



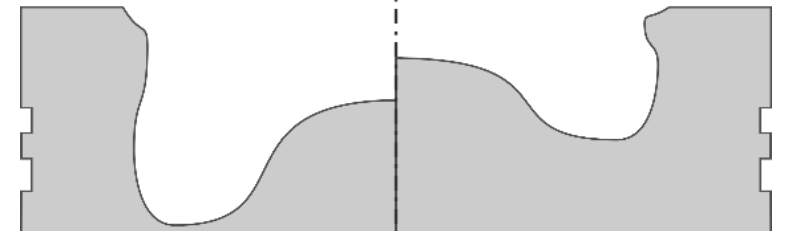
Bowl geometry #2



Bowl geometry #3



Bowl geometry #4



# CFD FOR APPLIED RESEARCH

## URANS coupled with optimization techniques

### ■ Noise optimization - GA optimization methodology

#### ➤ Solution convergence

- Reached after 725 simulations (25 citizens x 29 generations)

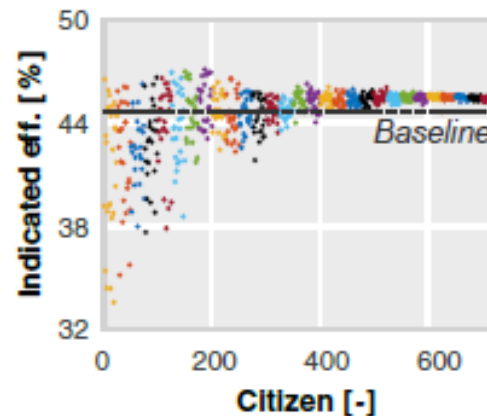
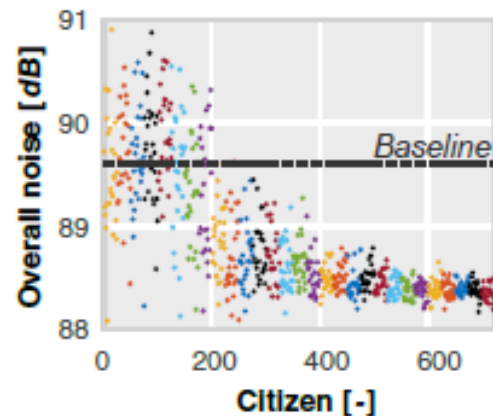
#### ➤ Synthesis of the results

##### • Constraints

- Pollutant levels are maintained below the predefined limits

##### • Objectives → clear improvements in:

- Noise level (~1.4 dB) → resonance energy ↓
- Efficiency (~0.8%) → fuel consumption ↓



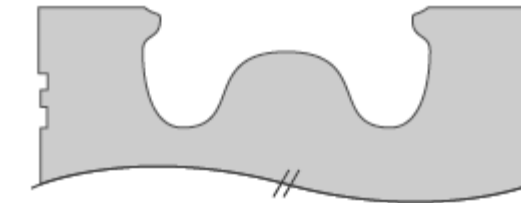
### Optimized configuration

Configuration	Baseline	Optimized
Spray included angle [deg]	75.0	83.4
Number of nozzle holes [-]	6	12
Swirl number [-]	1.26	1.69

#### Baseline bowl geometry



#### Optimized bowl geometry



Configuration	Baseline	Optimized
$E_{res}$ [kPa <sup>2</sup> ·s]	5.95	1.53
ISFC [g/kWh]	188.3	184.9
NO <sub>x</sub> [mg/s]	7.54	7.48
Soot [mg/s]	0.16	0.12
Overall noise [dB]	89.6	88.2
Indicated eff. [%]	44.7	45.5

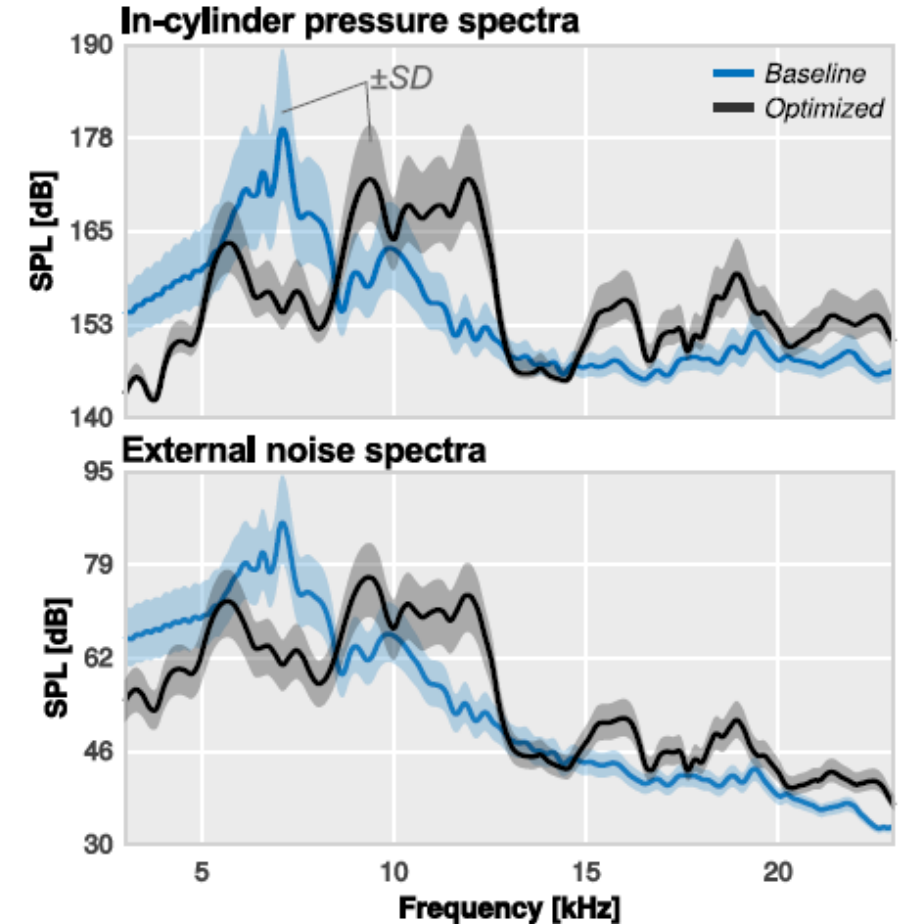
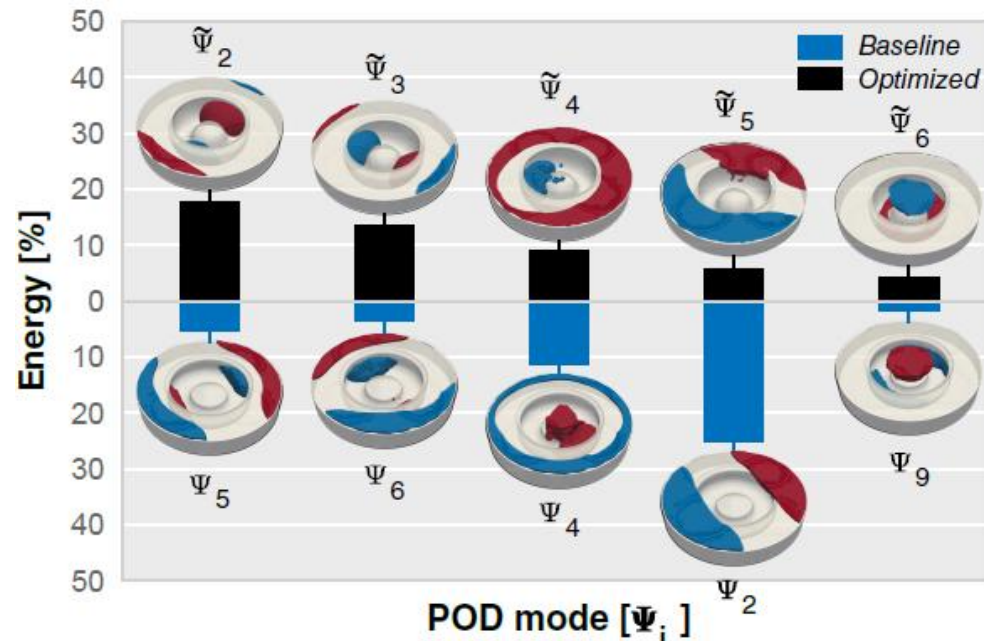
# CFD FOR APPLIED RESEARCH

## URANS coupled with optimization techniques

### ■ Noise optimization - GA optimization methodology

#### ➤ Acoustic energy distribution (FFT analysis)

- The acoustic energy is shifted towards higher frequencies
- Resonant modes experience a notable amplitude lowering
- Therefore, the overall resonant noise is reduced



## **CMT – CLEAN MOBILITY & THERMOFLUIDS**

**Address:** Edificio 6D, Universitat Politècnica de València  
Camino de Vera s/n, 46022 Valencia, Spain

**Phone:** +34963877650

**e-mail:** [cmt@mot.upv.es](mailto:cmt@mot.upv.es)

**Web:** [www.cmt.upv.es](http://www.cmt.upv.es)



# Thank you for your attention!

Josep Gomez-Soriano

[jogoso1@mot.upv.es](mailto:jogoso1@mot.upv.es)

[www.cmt.upv.es](http://www.cmt.upv.es)

[cmt@mot.upv.es](mailto:cmt@mot.upv.es)